9th INTERNATIONAL MARDIN ARTUKLU SCIENTIFIC RESEARCHES CONFERENCE

January 20–22, 2023, Mardin, TÜRKİYE

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BOUNDARY CONDITIONS DETERMINATION OF THE TANK MODEL AS A RIGID CYLINDRICAL SHELL WITH THE ELASTIC BOTTOM ON THE ELASTIC WINKLER BASE

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ABSTRACT

The tank model as a rigid cylindrical shell of radius R with the elastic bottom on the elastic Winkler base has been built. It has been considered the tank is partially filled with the ideal incompressible liquid to a height of H, Fig. 1. Let S_0 denote the liquid free surface, S_1 is the rigid cylindrical surface, and S_{bot} is the bottom elastic surface.

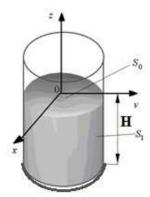


Figure 1. Cylindrical tank with the elastic bottom on the Winkler elastic base If the thickness h of a homogeneous plate is constant, then the motion equation of the plate in cylindrical coordinates has the form

$$D\Delta\Delta w + \rho_p h \underline{\hspace{1cm}}_{\partial t} 2 + Kw = q(r, \square, t). \tag{1}$$

Here D = 12_____($^E1^h_^3v_2$) is cylindrical stiffness, ρ_p is plate density, K – Winkler module, $q(r, \Box, t)$ – external force acting on the plate.

If the plate is in contact with the liquid, then

$$q(r, \square, t) = p(r, \square, t) + q_0(r, \square, t),$$

where p(x, y, t) is fluid pressure on the plate, $q_0(r, \Box, t)$ is disturbing force.

To find the pressure, it has been made the following assumptions: the liquid is ideal and incompressible, and its motion is vortex-free. Under these conditions, there is the velocity potential $\phi(x,y,z,t)$ that satisfies the Laplace equation

$$\frac{\partial 2 \square}{\partial z} \frac{\partial 2 \square}{\partial z} \frac{\partial 2 \square}{\partial z} = 0. \tag{2}$$

$$\frac{\partial x}{\partial z} \frac{\partial y}{\partial z} \frac{\partial z}{\partial z}$$

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The correlation between the velocity potential and the pressure determined from the linearized Cauchy-Lagrange integral [1-5]

$$\partial^{\square}$$

$$p - p_0 = -\rho_l [\underline{\hspace{1cm}}_{\partial t} + a_x(t)x + (g + a_z(t))\zeta],$$

where ρ_l is liquid density; p_0 is atmospheric pressure, $a_x(t)$, $a_z(t)$ are the acceleration components of the exciting force in horizontal and vertical directions, ζ – is the function describing the position and elevation level of the liquid free surface. Thus,

$$a_x(t) = a_h \cos \omega_h t$$
, $a_z(t) = a_v \cos \omega_v t$.

The boundary conditions for equation (1) are follows.

The no-flow condition has been fulfilled on the rigid cylindrical surface S₁

$$\frac{\partial \Box}{-} = 0.$$

$$\frac{\partial \mathbf{n} S_1}{\partial \mathbf{n}} S_1$$
(3)

On the elastic bottom, the no-flow condition takes the form

$$\frac{\partial \Box}{\partial \mathbf{n}} = \frac{\partial w}{\partial t}$$

$$\frac{\partial \mathbf{n}}{\partial t} = \frac{\partial w}{\partial t}$$
(4)

where w is deflection of the plate determined from equation (1) and the corresponding boundary conditions determined below [6-8].

On the free surface, the kinematic and dynamic boundary conditions in the form must be fulfilled

Here the function $\zeta = \zeta(t, x, y)$ characterizes the change in the level and free surface position over time, **n** is the external unit normal to the surface.

It has been used the boundary conditions for fixing the plate along the contour. The cylindrical coordinate system (r, \Box, z) has been applied. In the case of rigid fixation, there have been obtained the following boundary conditions:

$$w|r=R=0, dw_dr|r=R=0.$$
 (6)

The natural oscillations of the cylindrical shell – liquid system have been considered.

Thus, $q_0(r, \Box, t) = 0$, $a_x(t) = a_z(t) = 0$, and equation (1) takes the form

$$\partial^{2} w \qquad \partial \Box$$

$$D\Delta \Delta w + \rho_{p} h \underline{\qquad}_{\partial t} 2 + K w = -\rho_{l} \underline{\qquad}_{\partial t}. \tag{7}$$

Therefore, it is necessary to find the unknown functions w, \Box , ζ , that satisfy the system of differential equations

$$\partial^{2}w \qquad \partial \Box \qquad \partial^{2}\Box \ \partial^{2}\Box \ \partial^{2}\Box$$

$$D\Delta\Delta w + \rho_{p}h \underline{\qquad}_{\partial t}2 + Kw = -\rho_{l} \overline{\partial_{t}}, \quad \partial_{x}\overline{2 + \partial_{y}}2 + \partial_{z}2 = 0$$
(8)

and boundary conditions

 $\partial\Box$ $\partial\Box$ ∂w $\partial\Box$ $\partial\zeta$ dw

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 $\partial_{-}\mathbf{n}|S_1=0$, $\partial\mathbf{n}|S_{bot}=\underline{}\partial t$, $\partial\mathbf{n}|S_0=\partial t_{-}$, $p-p^0|S_0=0$, $w|^{r=R}=0$, $\underline{}dr|r=R=0$. (9) To obtain the unique solution of the system of equations (8) with boundary conditions (9), there have been added the Neumann condition

$$\iint \frac{\partial \Phi}{\partial t} dS_0 = 0_{S \partial \mathbf{n}} \tag{10}$$

П

The tank model has been built as the rigid cylindrical shell of radius R with the elastic bottom on the elastic Winkler base. Boundary conditions of the reservoir model have been defined.

Keywords: technogenic influence, hazardous liquid, seismic loads, storage tanks, petroleum products, sloshing.

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